

### CIGRÉ Generic Model for Single-Shaft Combined Cycle Power Plants

This model is located at system bus # \_\_\_\_\_ IBUS  
Machine # \_\_\_\_\_ I  
This model uses CONs starting with # \_\_\_\_\_ J  
and STATES starting with # \_\_\_\_\_ K  
and VARs starting with # \_\_\_\_\_ L  
and ICONs starting with # \_\_\_\_\_ M

CON	#	value	DESCRIPTION
J			$R_p$ (s) Electrical power feedback droop
J+1			$T_p$ (p.u.) Electrical power feedback time constant (>0)
J+2			$R_v$ (p.u.) Governor feedback droop
J+3			$K_{mwp}$ (p.u.) Proportional gain for outer loop MW control
J+4			$K_{mwi}$ (s) Integral gain for outer loop MW control
J+5			$rf_{max}$ (p.u.) Maximum limit on outer loop MW control loop
J+6			$rf_{min}$ (p.u.) Minimum limit on outer loop MW control loop
J+7			$dbd$ (p.u.) Intentional deadband
J+8			$err$ (p.u.) intentional error limit
J+9			$T_a$ (>0) (p.u.) Acceleration control time constant
J+10			$a_{set}$ (p.u.) Acceleration limit set-point (see note 2)
J+11			$K_{pg}$ (>0) Speed governor proportional gain (p.u.)
J+12			$K_{ig}$ (>0) Speed governor integral gain (p.u.)
J+13			$K_{dg}$ (p.u.) Speed governor derivative gain
J+14			$T_{dg}$ (p.u.) Speed governor derivative time constant
J+15			$K_{pa}$ (>0) Acceleration control proportional gain (p.u.)
J+16			$K_{ia}$ (p.u.) Acceleration control integral gain
J+17			$K_{pt}$ (>0) (p.u.) Temperature control proportional gain
J+18			$K_{it}$ (p.u.) Temperature control integral gain
J+19			$F_{max}$ (p.u.) Maximum fuel flow command
J+20			$F_{min}$ (p.u.) Minimum fuel flow command
J+21			$T_{limit}$ (p.u.) Temperature limit (see note 4)

CON	#	value	DESCRIPTION
J+22			$T_{thcp}$ (p.u.) Thermocouple time constant
J+23			$T_n$ (p.u.) Heat transfer lead time constant
J+24			$T_d$ (p.u.) Heat transfer lag time constant
J+25			$T_v$ (>0) (p.u.) Fuel system time constant
J+26			$V_{max}$ (p.u.) Maximum valve opening
J+27			$V_{min}$ (p.u.) Minimum valve opening
J+28			$F_m$ (p.u.) Fuel flow multiplier (see note 2)
J+29			$W_{fo}$ (p.u.) Full-speed no-load fuel flow
J+30			$K_t$ (>0) (p.u.) Turbine gain
J+31			$T_{tn1}$ (p.u.) Turbine numerator time constant 1
J+32			$T_{tn2}$ (p.u.) Turbine numerator time constant 2
J+33			$T_{td1}$ (p.u.) Turbine denominator time constant 1
J+34			$T_{td2}$ (>0) (p.u.) Turbine denominator time constant 2
J+35			$x_1 - 1^{st}$ point of turbine characteristic curve (see note 3)
J+36			$F(x_1)$
J+37			$x_2 - 2^{nd}$ point
J+38			$F(x_2)$
J+39			$x_3 - 3^{rd}$ point
J+40			$F(x_3)$
J+41			Trate – turbine rating (MW) (see note 1)

CON	#	value	DESCRIPTION
J+42			$P_{gt1}$ (p.u.) – point 1 (see notes 6 and 7)
J+43			$Q_{g1}$ (p.u.)
J+44			$P_{gt2}$ (p.u.) – point 2
J+45			$Q_{g2}$ (p.u.)
J+46			$P_{gt3}$ (p.u.) – point 3
J+47			$Q_{g3}$ (p.u.)
J+48			$P_{gt4}$ (p.u.) –point 4
J+49			$Q_{g4}$ (p.u.)
J+50			$P_{gt5}$ (p.u.) – point 5
J+51			$Q_{g5}$ (p.u.)
J+52			$P_{gt6}$ (p.u.) – point 6
J+53			$Q_{g6}$ (p.u.)
J+54			$P_{gt7}$ (p.u.) – point 7
J+55			$Q_{g7}$ (p.u.)
J+56			$T_{drum}$ (>0) – Drum time constant (s)
J+57			$K_m$ – Pressure loss due to flow friction in the boiler tubes (p.u.)
J+58			$T_{vs}$ (>0) – Actuator time constant for main steam (p.u.)
J+59			$K_p$ – Pressure controller proportional gain (p.u.)
J+60			$K_i$ – Pressure controller integral gain (p.u./s)
J+61			$T_n$ –Turbine lead time constant (s)
J+62			$T_d$ –Turbine lag time constant (s)
J+63			$Q_s$ – heat from sSupplemental firing (p.u.)
J+64			$B_v$ –Bypass valve opening (p.u.) (see notes 8 and 9)
J+65			$P_{ref}$ – Minimum steam pressure reference (p.u.)
J+66			$Trate_{ST}$ – ST rating (MW) (see note 5)

STATes	#	Description
K		Machine electrical power measurement
K+1		Governor differential control
K+2		Governor integral control
K+3		Turbine actuator
K+4		Turbine acceleration limiter integral control
K+5		Turbine load limiter measurement
K+6		Turbine load limiter integral control
K+7		Supervisor load control
K+8		Acceleration control
K+9		Temperature detection lead-lag
K+10		First state of turbine second order transfer function
K+11		Second state of turbine second order transfer function
K+12		Drum pressure ( $P_d$ )
K+13		turbine lead-lag block
K+14		pressure control integral channel
K+15		valve position (v)

VARs	#	Description
L		Lset Load Reference
L+1		Output of load limiter PI control
L+2		Output of governor PID control
L+3		Low value select ouptut
L+4		Output of acceleration limiter PI control
L+5		Pf
L+6		Supervisory load controller setpoint, MWset
L+7		Input of deadband block
L+8		output of deadband block
L+9		$P_g$
L+10		h (heat)
L+11		A - coefficient of F(x)
L+12		B – coefficient of F(x)
L+13		C – coefficient of F(x)
L+14		Steam flow through bypass valve ( $q_b$ )
L+15		Steam mass flow ( $q_t$ )
L+16		Turbine inlet pressure ( $P_t$ )
L+17		$Q_{imbalance}$
L+18		Power output from GT ( $P_{gt}$ )
L+19		Heat from GT exhaust ( $Q_e$ )
L+20		Calculated Bv

notes:

- 1) model adopts generator MVA base as turbine rating if parameter TRATE = 0.0
- 2) the parameter CON(J+10) =  $a_{set}$  should be set to +99 to disregard the acceleration limit control loop
- 3) the CIGRÉ document suggests implementing  $F(x)$  as a look-up table or a piecewise linear function, that would require specific manufacturer's data to be defined. The present implementation of this model assumes a quadratic approximation of the characteristics similar to the one given in Figure C-1 of the CIGRÉ document. The user should enter three points  $[x_i, F(x_i)]$  of the turbine characteristics curve and the model will determine the coefficients of a quadratic equation given as  $F(x) = A x^2 + B x + C$ . The characteristic in Figure C-1 can be approximated by using the following points: [0.95, 0.95575], [1.0, 1.001], and [1.05, 0.97568].
- 4) Tlimit corresponds to fuel demand required for 1 p.u. turbine power i.e.  $= (1/Kt + Wf_0)$ . Conversely, Tlimit determines the (steady state) maximum power output of the turbine:  $P_{max} = (Tlimit - Wf_0) \times Kt$ , expressed in per unit of turbine rating Trate.
- 5) model adopts generator MVA base as steam turbine rating if parameter TRATE<sub>ST</sub> = 0.0
- 6) Points in the look-up table should have  $Q_g$  greater or equal to zero and should monotonically increase. Once a decrease is detected, no more points are considered in the look-up table. Thus, the user can enter [0.0, 0.0] for those points not needed.
- 7) If less than 7 points are required in the look-up table, the remainder data points should be set to [0.0, 0.0]. The heat  $Q_g$  is considered constant and equal to the last value  $Q_{gi}$  when  $P_{gt}$  is greater than  $P_{gti}$ . **Heat is not extrapolated beyond the last point in the look-up table.**
- 8) The bypass valve opening  $B_v$  should be a number between 0 and 1. If a negative value is entered, the model will calculate the bypass valve opening to minimize any imbalances in heat
- 9) Setting  $B_v$  to 0.0 corresponds to no steam bypass and all available steam is used for electric power production
- 10) This model was proposed in CIGRÉ Technical Brochure on Modeling of Gas Turbines and Steam Turbines in Combined-Cycle Power Plants, Task Force 25 of Advisory Group 02 of Study Committee 38, April 2003.

a) initialization in sliding mode

$$\begin{cases} q_t = P_{st} = P_t \\ q_b = B_v \sqrt{P_t} \Rightarrow Q_g = P_{st} + B_v \sqrt{P_{st}} - Q_s \\ Q_g = q_t + q_b - Q_s \end{cases}$$

$$Q_g = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} (P_{gt} - P_{gt_A}) + Q_{g_A} = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$P_{st} + B_v \sqrt{P_{st}} - Q_s = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$P_{st} \cdot T_{rate_{st}} + P_{gt} \cdot T_{rate_{gt}} = P_{mech} \cdot M_{base} \Rightarrow P_{gt} = \frac{P_{mech} \cdot M_{base} - P_{st} \cdot T_{rate_{st}}}{T_{rate_{gt}}}$$

$$P_{st} + B_v \sqrt{P_{st}} - Q_s = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} \left[ \frac{P_{mech} \cdot M_{base} - P_{st} \cdot T_{rate_{st}}}{T_{rate_{gt}}} \right] + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$\left[ 1 + \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} \frac{T_{rate_{st}}}{T_{rate_{gt}}} \right] P_{st} + B_v \sqrt{P_{st}} = Q_s + \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{mech} \frac{M_{base}}{T_{rate_{gt}}} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

b) if  $P_t < P_{ref} \Rightarrow$  pressure control  $\Rightarrow P_t = P_{ref}$

$$\begin{cases} q_t = P_{st} = P_{ref} \cdot v \\ q_b = B_v \sqrt{P_{ref}} \Rightarrow Q_g = P_{st} + B_v \sqrt{P_{ref}} - Q_s \\ Q_g = q_t + q_b - Q_s \end{cases}$$

$$Q_g = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} (P_{gt} - P_{gt_A}) + Q_{g_A} = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$P_{st} + B_v \sqrt{P_{ref}} - Q_s = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$P_{st} \cdot T_{rate_{st}} + P_{gt} \cdot T_{rate_{gt}} = P_{mech} \cdot M_{base} \Rightarrow P_{gt} = \frac{P_{mech} \cdot M_{base} - P_{st} \cdot T_{rate_{st}}}{T_{rate_{gt}}}$$

$$P_{st} + B_v \sqrt{P_{ref}} - Q_s = \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} \left[ \frac{P_{mech} \cdot M_{base} - P_{st} \cdot T_{rate_{st}}}{T_{rate_{gt}}} \right] + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right)$$

$$\left[ 1 + \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} \frac{T_{rate_{st}}}{T_{rate_{gt}}} \right] P_{st} = Q_s + \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{mech} \frac{M_{base}}{T_{rate_{gt}}} + \left( Q_{g_A} - \frac{Q_{g_B} - Q_{g_A}}{P_{gt_B} - P_{gt_A}} P_{gt_A} \right) - B_v \sqrt{P_{ref}}$$

